

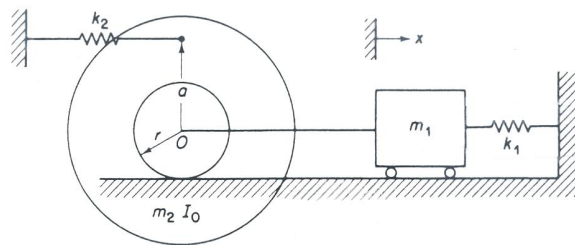
# HW 5, ME 440 Intermediate Vibration, Fall 2017

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## Problem 1

The stepped cylinder is connected to a spring of stiffness  $k_2$  and an inextensible cable. The other end of the inextensible cable is attached to mass  $m_1$ . The stepped cylinder rolls without slip on the fixed surface. The mass  $m_1$  rolls on 2 massless cylinders. Assume the system will be limited to small displacements. The total mass of the stepped cylinder is  $m_2$  and its mass moment of inertia about point  $O$  is  $I_0$ .



- In preparation for using Newton's Second Law, sketch the free-body diagram(s) **and** inertial diagram for this system.
- Using Newton's Laws exclusively, determine the differential equation of motion for small angular oscillations of the mass  $m_1$  (in terms of the generalized coordinate  $x$ ).

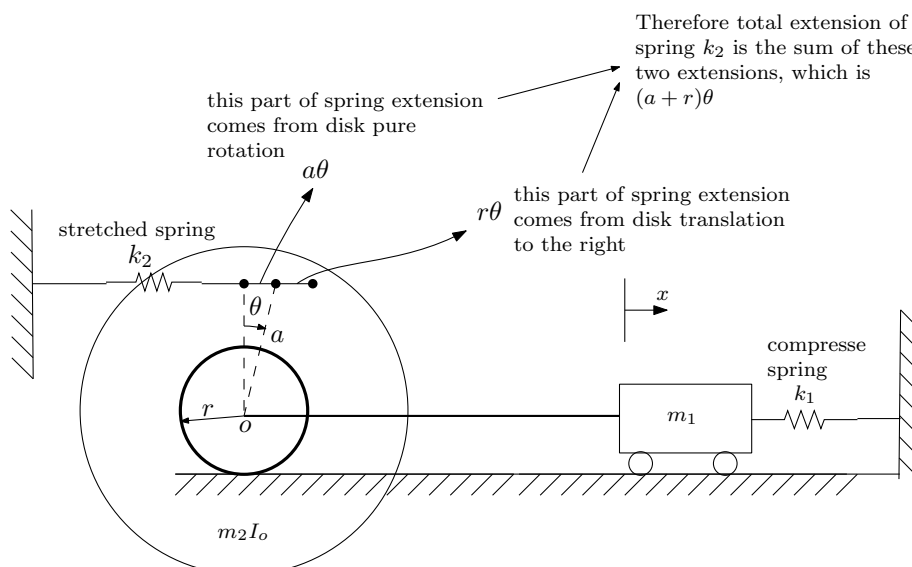
## Problem 2

Repeat Problem 1 but use  $T_{max} = U_{max}$  to find the natural frequency of the system.

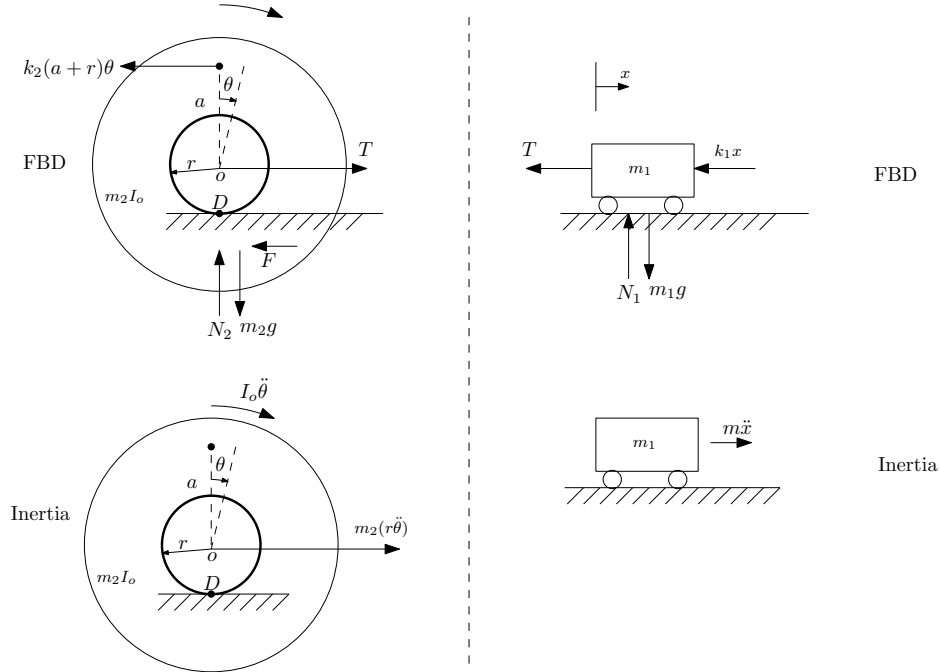
## 0.1 Problem 1

### 0.1.1 Part (A)

We start by assuming motion to the right, such that the small disk  $m_2$  rotates clockwise as shown below. So the  $k_2$  spring is stretched by amount  $a\theta$  which come due to pure rotation, and it also stretch by  $r\theta$  due to disk translation to the right at same time, therefore the spring  $k_1$  will stretch by amount  $(a + r)\theta$  and the  $k_1$  spring will be compressed by amount  $x$ .



Based on the above, the following is the free body diagram for  $m_2$  and  $m_1$  and the corresponding kinematic diagrams. This assumes small angle  $\theta$  and that springs remain straight.



### 0.1.2 Part (B)

Since cable is inextensible, then the constraint is that  $x = r\theta$ . Starting from the FBD for  $m_1$

$$\begin{aligned} \sum F_x &= m_1 \ddot{x} \\ -T - k_1 x &= m_1 \ddot{x} \\ m_1 \ddot{x} + k_1 x &= -T \end{aligned} \quad (1)$$

We do not need to resolve forces in vertical direction, since no motion is in that direction. To find  $T$ , which is the tension in cable, we go back to  $m_2$  and find  $T$ .

We can do this part in two ways, either by taking moments around the instantaneous center of zero velocity which is point  $D$  at bottom of the small cylinder shown in the diagram, or we can take moments around the C.M. of the disk and then use another equation to solve for the friction  $F$ . We will show both methods, and that they give the same result.

Method one, using instantaneous center of zero velocity

Take moments around point  $D$  as shown in figure in order to not have to account for the friction force  $F$  and the  $N_2$  force on  $m_2$  and using positive as anti-clockwise gives

$$\begin{aligned} \sum M_D &= -I_D \ddot{\theta} \\ k_2(a+r)\theta(a+r) - Tr &= -\overbrace{(I_o + m_2 r^2)}^{\text{parallel axes}} \ddot{\theta} \\ T &= \frac{k_2(a+r)^2 \theta + (I_o + m_2 r^2) \ddot{\theta}}{r} \end{aligned}$$

But due to constraint, then  $\theta = \frac{x}{r}$ ,  $\ddot{\theta} = \frac{\ddot{x}}{r}$ . Hence the above can be written as

$$\begin{aligned} T &= \frac{k_2 \frac{x}{r} (a+r)^2 + (I_o + m_2 r^2) \frac{\ddot{x}}{r}}{r} \\ &= \frac{x k_2 (a+r)^2}{r^2} + \frac{(I_o + m_2 r^2) \ddot{x}}{r^2} \end{aligned} \quad (2)$$

Substituting (2) into (1) gives

$$m_1\ddot{x} + k_1x = -\left(\frac{xk_2(a+r)^2}{r^2} + \frac{(I_o + m_2r^2)\ddot{x}}{r^2}\right)$$

$$m_1\ddot{x} + \frac{(I_o + m_2r^2)\ddot{x}}{r^2} + k_1x + \frac{xk_2(a+r)^2}{r^2} = 0$$

$$\ddot{x}\left(m_1 + \frac{(I_o + m_2r^2)}{r^2}\right) + x\left(k_1 + \frac{k_2(a+r)^2}{r^2}\right) = 0$$

$$\ddot{x}\left(\frac{m_1r^2 + (I_o + m_2r^2)}{r^2}\right) + x\left(\frac{k_1r^2 + k_2(a+r)^2}{r^2}\right) = 0$$

Hence

$$\ddot{x}(m_1r^2 + (I_o + m_2r^2)) + x(k_1r^2 + k_2(a+r)^2) = 0$$

In standard form

$$\ddot{x} + x\frac{k_1r^2 + k_2(a+r)^2}{r^2(m_1 + m_2) + I_o} = 0 \quad (3)$$

Or

$$\ddot{x} + \omega_n^2x = 0$$

Where

$$\omega_n^2 = \frac{r^2k_1 + k_2(a+r)^2}{r^2(m_1 + m_2) + I_o}$$

Method two, moments around center of mass

Using this method. We start by taking moments around the center of mass of the disk  $m_2$  and using positive as anti-clockwise gives

$$\begin{aligned} \sum M_o &= -I_o\ddot{\theta} \\ (k_2(a+r)\theta)a - Fr &= -I_o\ddot{\theta} \\ F &= \frac{1}{r}(I_o\ddot{\theta} + (k_2(a+r)\theta)a) \end{aligned} \quad (4)$$

Now resolving forces in the  $x$  direction for  $m_2$ , gives (with positive to the right)

$$\begin{aligned} \sum F_x &= m_2r\ddot{\theta} \\ T - k_2(a+r)\theta - F &= m_2r\ddot{\theta} \end{aligned} \quad (5)$$

Plugging (4) into (5) gives  $T$

$$T - k_2(a+r)\theta - \frac{1}{r}(I_o\ddot{\theta} + (k_2(a+r)\theta)a) = m_2r\ddot{\theta}$$

Solving for  $T$  gives

$$T = m_2r\ddot{\theta} + \frac{1}{r}(I_o\ddot{\theta} + (k_2(a+r)\theta)a) + k_2(a+r)\theta$$

We now use the constraint that  $x = r\theta$  to write everything in  $x$ . Hence  $\theta = \frac{x}{r}$ ,  $\ddot{\theta} = \frac{\ddot{x}}{r}$  and the above now becomes

$$\begin{aligned} T &= m_2r\frac{\ddot{x}}{r} + \frac{1}{r}\left(I_o\frac{\ddot{x}}{r} + \left(k_2(a+r)\frac{x}{r}\right)a\right) + k_2(a+r)\frac{x}{r} \\ &= m_2\ddot{x} + \frac{1}{r^2}(I_o\ddot{x} + (k_2(a+r)x)a) + k_2(a+r)\frac{x}{r} \end{aligned}$$

Now that we found  $T$ , we go back to the equation of motion for  $m_1$  in (1) and substitute the above into it, the result becomes

$$\begin{aligned} m_1\ddot{x} + k_1x &= -T \\ &= -\left(m_2\ddot{x} + \frac{1}{r^2}(I_o\ddot{x} + (k_2(a+r)x)a) + k_2(a+r)\frac{x}{r}\right) \end{aligned}$$

Collecting terms

$$\begin{aligned} \ddot{x} \left( m_1 + m_2 + \frac{I_o}{r^2} \right) + k_1 x + \frac{1}{r^2} ((k_2 (a+r)x)a) + k_2 (a+r) \frac{x}{r} &= 0 \\ \ddot{x} \left( m_1 + m_2 + \frac{I_o}{r^2} \right) + x \left( k_1 + \frac{1}{r^2} (k_2 (a+r)a) + k_2 (a+r) \frac{1}{r} \right) &= 0 \\ \ddot{x} \left( m_1 + m_2 + \frac{I_o}{r^2} \right) + x \left( k_1 + \frac{k_2}{r^2} [(a+r)a + r(a+r)] \right) &= 0 \\ \ddot{x} \left( m_1 + m_2 + \frac{I_o}{r^2} \right) + x \left( k_1 + \frac{k_2}{r^2} [a^2 + ra + ar + r^2] \right) &= 0 \\ \ddot{x} \left( m_1 + m_2 + \frac{I_o}{r^2} \right) + x \left( k_1 + \frac{k_2}{r^2} [a^2 + 2ar + r^2] \right) &= 0 \\ \ddot{x} \left( m_1 + m_2 + \frac{I_o}{r^2} \right) + x \left( k_1 + \frac{k_2}{r^2} (a+r)^2 \right) &= 0 \end{aligned}$$

Or

$$\begin{aligned} \ddot{x} (r^2 (m_1 + m_2) + I_o) + x (r^2 k_1 + k_2 (a+r)^2) &= 0 \\ \ddot{x} + x \frac{r^2 k_1 + k_2 (a+r)^2}{r^2 (m_1 + m_2) + I_o} &= 0 \end{aligned}$$

Which is the same equation of motion found in the first method.

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## 0.2 Problem 2

In Rayleigh energy method, we ignore any friction, and assume motion is simple harmonic motion (which is valid, since there is no damping).

The Kinetic energy  $T$  of the system is (since disk rolls with no slip)

$$T = \overbrace{\frac{1}{2} I_o \dot{\theta}^2}^{\text{disk}} + \overbrace{\frac{1}{2} m_2 v_{cg}^2}^{\text{cart}} + \frac{1}{2} m_1 \dot{x}^2$$

But  $v_{cg} = r\dot{\theta}$ , hence the above becomes

$$T = \frac{1}{2} I_o \dot{\theta}^2 + \frac{1}{2} m_2 (r\dot{\theta})^2 + \frac{1}{2} m_1 \dot{x}^2$$

But due to constraint, then  $\theta = \frac{x}{r}$ , then  $\dot{\theta} = \frac{\dot{x}}{r}$  and the above becomes

$$\begin{aligned} T &= \frac{1}{2} I_o \left( \frac{\dot{x}}{r} \right)^2 + \frac{1}{2} m_2 \left( r \frac{\dot{x}}{r} \right)^2 + \frac{1}{2} m_1 \dot{x}^2 \\ &= \frac{1}{2} I_o \frac{\dot{x}^2}{r^2} + \frac{1}{2} m_2 \dot{x}^2 + \frac{1}{2} m_1 \dot{x}^2 \\ &= \frac{1}{2} \dot{x}^2 \left( \frac{I_o}{r^2} + m_2 + m_1 \right) \end{aligned} \tag{1}$$

The potential energy is

$$\begin{aligned} U &= \frac{1}{2} k_2 ((a+r)\theta)^2 + \frac{1}{2} k_1 x^2 \\ &= \frac{1}{2} k_2 \left( (a+r) \frac{x}{r} \right)^2 + \frac{1}{2} k_1 x^2 \\ &= \frac{1}{2} k_2 (a+r)^2 \frac{x^2}{r^2} + \frac{1}{2} k_1 x^2 \end{aligned} \tag{2}$$

To find  $T_{\max}$  and  $U_{\max}$ , we now assume  $m_1$  undergoes simple harmonic motion given by  $x(t) = X_{\max} \sin(\omega_n t)$ . Hence  $\dot{x} = X_{\max} \omega_n \cos \omega_n t$ . Therefore

$$\dot{x}_{\max} = X_{\max} \omega_n$$

$$x_{\max} = X_{\max}$$

Therefore using these into (1) and (2) gives

$$T_{\max} = \frac{1}{2} (\dot{x}_{\max})^2 \left( \frac{I_o}{r^2} + m_2 + m_1 \right)$$

$$U_{\max} = \frac{1}{2} k_2 (a+r)^2 \frac{x_{\max}^2}{r^2} + \frac{1}{2} k_1 x_{\max}^2$$

Or

$$T_{\max} = \frac{1}{2} (X_{\max} \omega_n)^2 \left( \frac{I_o}{r^2} + m_2 + m_1 \right)$$

$$U_{\max} = \frac{1}{2} X_{\max}^2 \left( \frac{k_2 (a+r)^2}{r^2} + k_1 \right)$$

Hence

$$T_{\max} = U_{\max}$$

$$\frac{1}{2} (X_{\max} \omega_n)^2 \left( \frac{I_o}{r^2} + m_2 + m_1 \right) = \frac{1}{2} X_{\max}^2 \left( \frac{k_2 (a+r)^2}{r^2} + k_1 \right)$$

$$\omega_n^2 \left( \frac{I_o}{r^2} + m_2 + m_1 \right) = \frac{k_2 (a+r)^2 + r^2 k_1}{r^2}$$

Solving for  $\omega_n^2$

$$\omega_n^2 = \frac{k_2 (a+r)^2 + r^2 k_1}{I_o + r^2 (m_2 + m_1)}$$

Therefore the equation of motion for  $m_2$  is

$$\ddot{x} + \omega_n^2 x = 0$$

$$\ddot{x} + \frac{k_2 (a+r)^2 + r^2 k_1}{I_o + r^2 (m_2 + m_1)} x = 0$$

Comparing this to the solution found in first problem, we see they are the same. The Rayleigh energy method was much simpler in this case. But we have to ignore any friction, and assume motion is harmonic, which is reasonable, since this is single degree of freedom system.